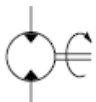

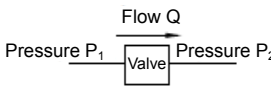

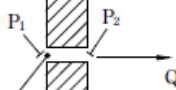
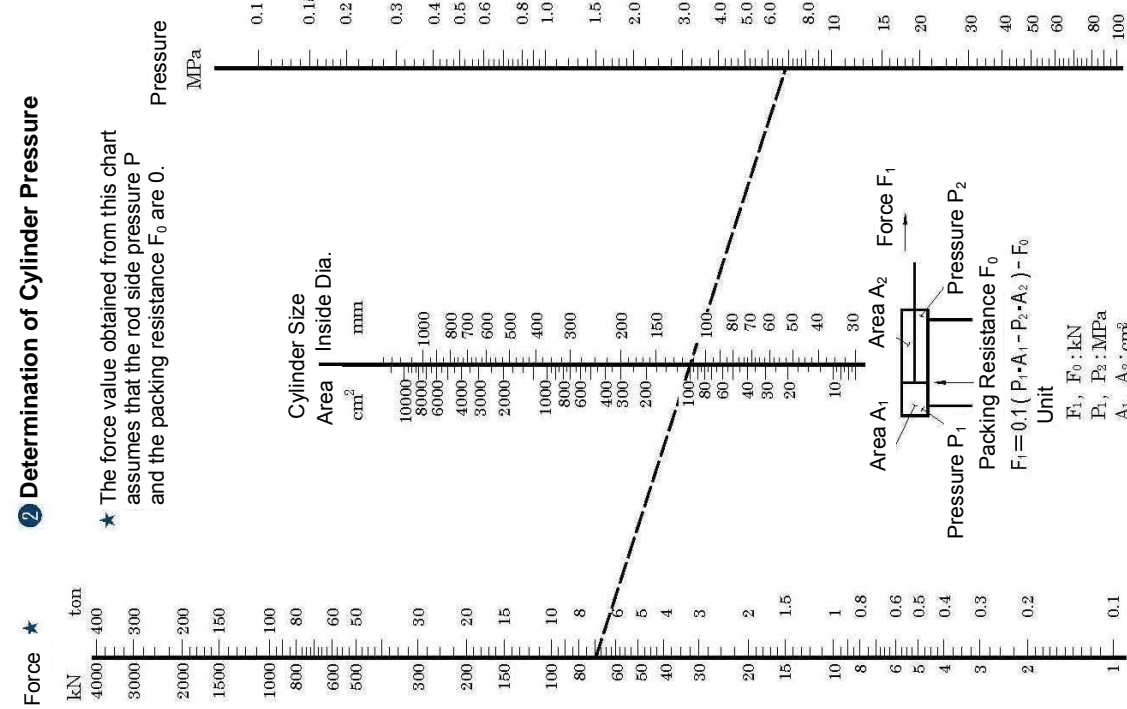
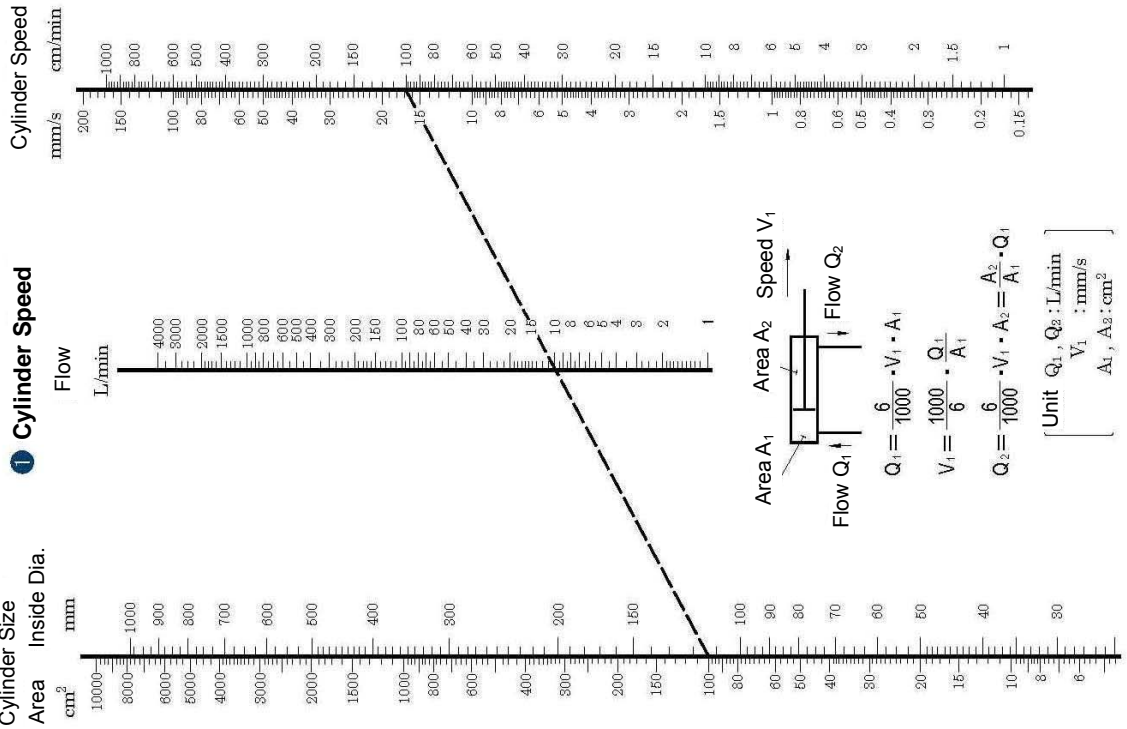
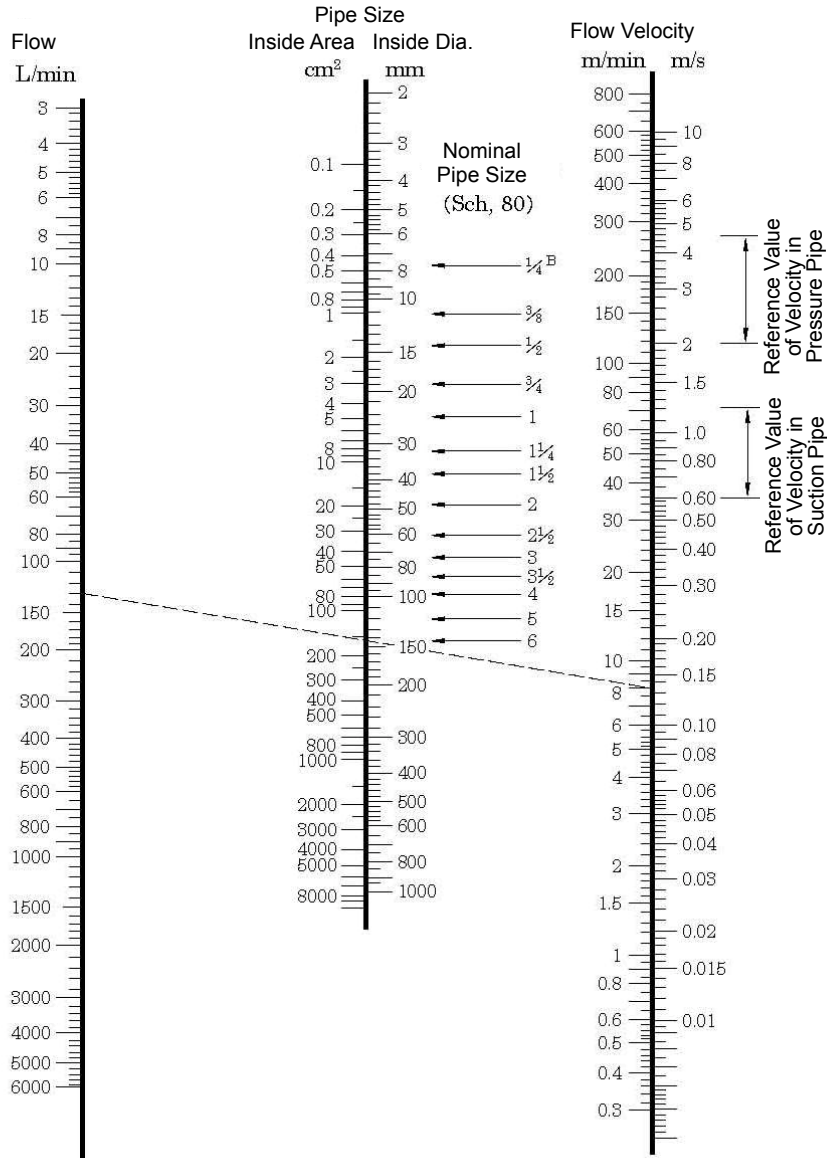


	SI Unit	Engineering Unit (Reference)
Hydraulic Pump	<p>● Hydraulic Power (Pump Output)</p> $L_o = \frac{P \cdot Q}{60}$ <p>L_o: Hydraulic Power kW P: Pressure MPa Q: Flow L/min * 1 kW = 1 kN·m/s = 60 kN·m/min</p>	<p>$L_o = \frac{P \cdot Q}{612}$</p> <p>$L_o$: Hydraulic Power kW P: Pressure kgf/cm² Q: Flow L/min * 1 kW = 102 kgf·m/s = 6120 kgf·m/min</p>
	<p>● Shaft Input</p> $L_i = \frac{2\pi TN}{60000}$ <p>L_i: Shaft Input kW T: Shaft Torque N·m N: Shaft Speed r/min</p>	<p>$L_i = \frac{2\pi TN}{6120}$</p> <p>$L_i$: Shaft Input kW T: Shaft Torque kgf·m N: Shaft Speed rpm</p>
	<p>● Volumetric Efficiency</p> $\eta_v = \frac{Q_p}{Q_o} \times 100$ <p>η_v: Volumetric Efficiency % Q_p: Flow at Pressure P L/min Q_o: Flow at No Load L/min * Q_o - Q_p = Pump's Total Internal Leakage</p>	
	<p>● Overall Efficiency</p> $\eta = \frac{L_o}{L_i} \times 100 = \frac{P \cdot Q}{60 L_i} \times 100$ <p>η: Overall Efficiency % L_o: Hydraulic Power kW L_i: Shaft Input kW P: Discharge Pressure MPa Q: Flow L/min</p>	<p>$\eta = \frac{L_o}{L_i} \times 100 = \frac{P \cdot Q}{612 L_i} \times 100$</p> <p>$\eta$: Overall Efficiency % L_o: Hydraulic Power kW L_i: Shaft Input kW P: Discharge Pressure kgf/cm²</p>
<p>● Hydraulic Motor Output</p>  $L = \frac{2\pi T \cdot N}{60000}$ <p>L: Output kW T: Torque Nm N: Speed r/min</p>	<p>$L = \frac{2\pi T \cdot N}{6120}$</p> <p>L: Output kW T: Torque kgf·m N: Speed rpm</p>	
<p>● Cylinder Output</p>  $L = \frac{F \cdot V}{60}$ <p>L: Output kW F: Force kN V: Speed m/min</p>	<p>$L = \frac{F \cdot V}{6120}$</p> <p>L: Output kW F: Force kgf V: Speed m/min</p>	
<p>● Valve Power Loss</p>  <p>Pressure Loss: $\Delta P = P_1 - P_2$ Power Loss between Valve Inlet and Outlet: L</p> $L = \frac{\Delta P \cdot Q}{60}$ <p>L: kW ΔP: MPa Q: L/min</p>	<p>$L = \frac{\Delta P \cdot Q}{612}$</p> <p>L: kW ΔP: kgf/cm² Q: L/min</p>	
<p>● Viscosity (Absolute) and Kinematic Viscosity</p> $\mu = \rho \cdot \nu_1 = \rho \cdot \nu_2 \times 10^{-6}$ <p>μ: Viscosity (Absolute) Pa·s (= N·s/m²) ρ: Density kg/m³ ν_1: Kinematic Viscosity m²/s ν_2: Kinematic Viscosity mm²/s</p>	<p>$\mu = \rho \cdot \nu_1 = \frac{\gamma}{g} \cdot \nu_1 = \frac{\gamma \cdot \nu_2}{100g}$</p> <p>$\mu$: Viscosity (Absolute) kgf·s/cm² ρ: Density kgf·s²/cm⁴ ν_1: Kinematic Viscosity cm²/s ν_2: Kinematic Viscosity cSt γ: Specific Gravity kgf/cm³ g: Gravitational Acceleration 980 cm/s² * 1 cSt = 0.01 cm²/s</p>	
<p>● Reynolds Number</p>  <p>R: Reynolds Number ν: Kinematic Viscosity</p> $R = \frac{V \cdot d}{\nu_1} = \frac{4000Q}{60\pi d \cdot \nu_1} = \frac{2120Q}{d \cdot \nu_2}$ <p>R: Dimensionless V: cm/s d: cm ν_1: cm²/s ν_2: mm²/s [cSt] Q: L/min</p> <p>* R < 2300: Laminar Flow R > 2300: Turbulent Flow</p>		
<p>● Orifice Flow</p>  <p>A: Opening Area $\Delta P = P_1 - P_2$ C = Discharge Coefficient γ = Specific Gravity ρ = Density</p> $Q = C \cdot A \sqrt{\frac{2\Delta P}{\rho}} \times 10^6 \times 6$ <p>Q: L/min C: Dimensionless Discharge Coefficient ΔP: MPa ρ: kg/m³ A: cm²</p>	<p>$Q = C \cdot A \sqrt{\frac{2g}{\gamma} \Delta P} \times \frac{60}{1000} = 2.66 C \cdot A \sqrt{\frac{\Delta P}{\gamma}}$</p> <p>Q: L/min C: Dimensionless Discharge Coefficient γ: kgf/cm³ ΔP: kgf/cm² A: cm²</p>	
<p>(Note) The value of discharge coefficient depends on the flow channel geometry and the Reynolds number; it generally ranges from 0.6 to 0.9.</p>		



1 Pipe Size/ Flow Velocity



2 Steel Tubes/Pipes SGP, STS370, STPS2 Carbon Steel Pipes

Pipe Type ->		SGP (JIS G 3452)		STS370 (JIS G 3455)												
Nominal Pres. MPa ->		2		4		6		10		16		25		35		
Safety Factor ->		8 or more		6 or more		5 or more		4 or more								
Nominal Dia. (A)	Outside (B)	Thickness mm	Thick mm	Sch. No.	Thick mm	Sch. No.	Thick mm	Sch. No.	Thick mm	Sch. No.	Thick mm	Sch. No.	Thick mm	Sch. No.	Thick mm	Sch. No.
8	1/4	13.8													3.0	80
10	3/8	17.3													3.2	80
15	1/2	21.7				2.8	40								4.7	160
20	3/4	27.2				2.9	40						3.7	80	5.5	160
25	1	34.0				3.4	40	4.5	80				6.4	160	6.4	160
32	1 1/4	42.7				3.6	40	4.9	80				6.4	160	8.0	★
40	1 1/2	48.6				3.7	40	5.1	80				7.1	160	9.0	★
50	2	60.5				3.9	40	5.5	80				8.7	160	11.2	★
65	2 1/2	76.3	4.2	5.2	40			7.0	80	9.5	160				14.2	★
80	3	89.1	4.2	5.2	40			7.6	80	11.1	160				16.5	★
90	3 1/2	101.6	4.2	5.7	40	8.1	80			12.7	160				20.0	★
100	4	114.3	4.5	6.0	40	8.6	80			13.5	160				20.0	★
125	5	139.8	4.5	9.5	80			15.9	160							
150	6	165.2	5.0	11.0	80			18.2	160							

Precision Carbon Steel Tubes for Compression Type Tube Fittings Thickness (mm)

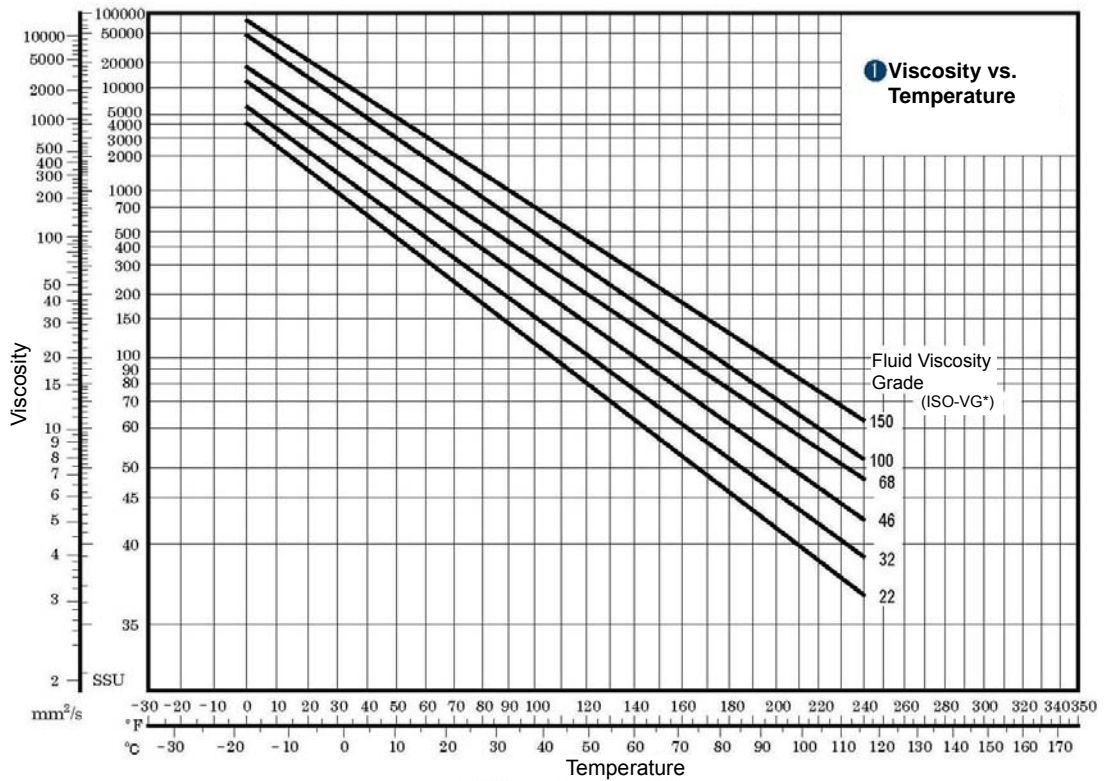
Outside mm	Safety Factor	Nominal Pres. MPa			
		10	16	25	35
6					1.5
10				1.5	2.0
12				2.0	2.5
16		2.0		3.0	
20		2.0	2.5	3.0	
25		2.5		4.0	

Note)

- STPS2 defined in JIS B 2351-1 Annex 2.
- For selection considerations, refer to Note 1 in the "Carbon Steel Pipes" section.
- Designation (Example) STPS2-12 × 2.5

Note)

- The selection of steel pipes based on the operating pressure may be difficult, since the pressure fluctuation, pipe vibration, pipe connection type, and other factors must be considered. Refer to the nominal pressure values and their corresponding safety factors in the left table for pipe selection.
- "Sch. No." is an abbreviation for schedule number. Note that "★" indicates special thick wall steel pipes with no schedule number.
<Reference>
JIS G 3452, 3454 to 64
Description
Schedule number = 10 × P/S where
P: Operating pressure MPa
S: Allowable stress MPa
- Designation (-B Series of Yuken)
(Example 1)
SGP pipe: SGP-2 1/2B
(Example 2)
STS370 with Sch. No.: STS370-3/4B × Sch. 80
(Example 3)
STS370 special thick wall steel pipe: STS370-1 1/4B × 8.0 t



② Viscosity Conversion Chart

Use the following equations when the viscosity is 100 mm²/s or more.

SSU × 0.220 = mm²/s
 RSS × 0.2435 = mm²/s
 °E × 7.6 = mm²/s

